

Example 2: Flow in Circular Pipe

(Set M = Momentum ($\text{kg}\cdot\text{m}/\text{s}^2$), δ = Thickness of film (m), V_z = Velocity (m/s) and P = Pressure (N/m^2)

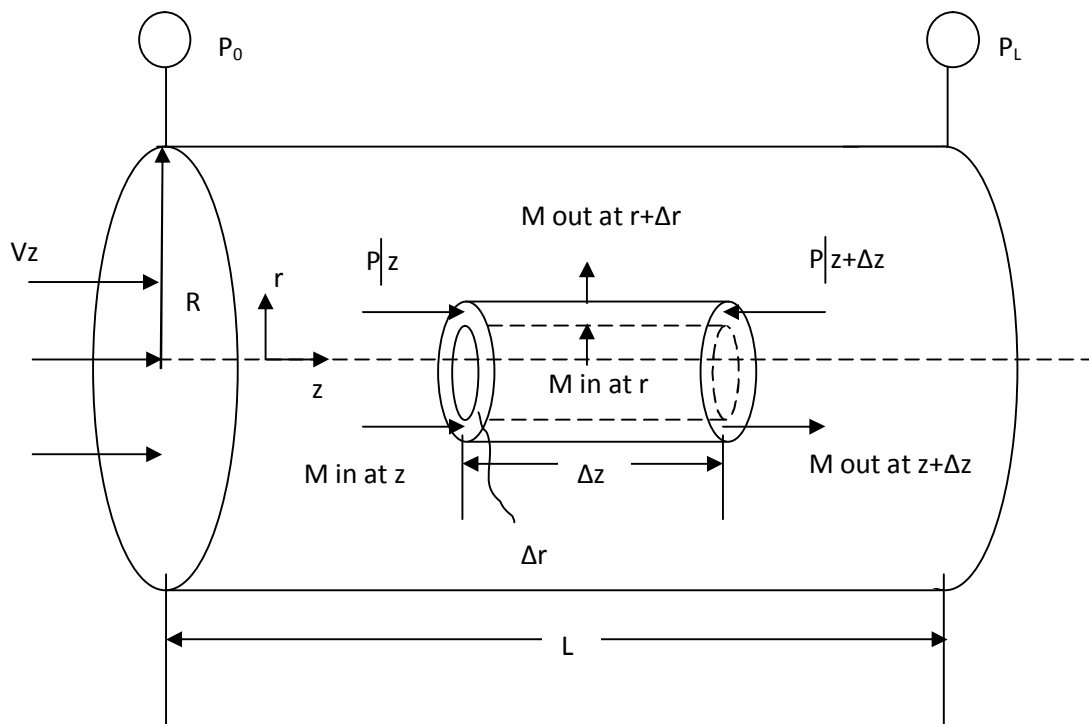


Fig. 2 Flow in circular pipe in horizontal direction

Apply general law of conservation momentum (force balance)

$$\begin{array}{ccc}
 \text{Term 1} & \text{Term 2} & \text{Term 3} \\
 \left(\begin{array}{l} \text{Rate of momentum in} \\ \text{Shell in z-direction} \\ \text{by convection} \end{array} \right) & - \left(\begin{array}{l} \text{Rate of momentum} \\ \text{out shell in } z+\Delta z \\ \text{by convection} \end{array} \right) & + \left(\begin{array}{l} \text{Rate of momentum} \\ \text{in shell in r-direction} \\ \text{by molecular diffusion} \end{array} \right) \\
 \text{Term 4} & \text{Term 5} & \text{Term 6} \\
 - \left(\begin{array}{l} \text{Rate of momentum} \\ \text{out shell in } r+\Delta r \\ \text{By molecular diffusion} \end{array} \right) & + \left(\begin{array}{l} \text{Sum of external} \\ \text{forces acting} \\ \text{on shell} \end{array} \right) & \left(= \text{Rate of momentum} \right. \\
 & & \left. \text{accumulation} \right)
 \end{array}$$

For term 6 = 0 (assume steady state system)

For term 1: Rate of momentum in shell at $z = z$ by convection

$$(\text{mass flow rate})(v_z) \Big|_{z=z} = (\rho \cdot v_z \cdot 2\pi r \cdot \Delta r) v_z \Big|_{z=z}$$

For term 2: Rate of momentum out shell at $z = z + \Delta z$ by convection

$$(\text{mass flow rate})(v_z) \Big|_{z=z+\Delta z} = (\rho \cdot v_z \cdot 2\pi r \cdot \Delta r) v_z \Big|_{z=z+\Delta z}$$

For term 3: Rate of momentum in shell at $r = r$ by diffusion

$$(2\pi r \cdot \Delta r) \tau_{rz} \Big|_{r=r}$$

For term 4: Rate of momentum out shell at $r = r + \Delta r$

$$(2\pi r \cdot \Delta r) \tau_{rz} \Big|_{r=r+\Delta r}$$

For term 5: Sum of external forces acting on shell

$$= \underbrace{(2\pi r \cdot \Delta r) \cdot P \Big|_{z=z} - (2\pi r \cdot \Delta r) \cdot P \Big|_{z=z+\Delta z}} + \underbrace{\text{gravitational force}} = 0$$

Be careful with the sign of each term.

Because the flow is in horizontal, no gravitational force

Combine all terms

So, from momentum balance around shell

$$(\rho \cdot v_z \cdot 2\pi r \cdot \Delta r) v_z \Big|_z - (\rho \cdot v_z \cdot 2\pi r \cdot \Delta r) v_z \Big|_{z+\Delta z} + (2\pi r \cdot \Delta z) \tau_{rz} \Big|_r - (2\pi r \cdot \Delta z) \tau_{rz} \Big|_{r+\Delta r} + (2\pi r \cdot \Delta r) P \Big|_z - (2\pi r \cdot \Delta r) P \Big|_{z+\Delta z} = 0$$

Divided through by $2\pi \Delta r \Delta z$

$$-\left(\frac{r \rho v_z^2 \Big|_{z+\Delta z} - r \rho v_z^2 \Big|_z}{\Delta z} \right) - \left(\frac{r \tau_{rz} \Big|_{r+\Delta r} - r \tau_{rz} \Big|_r}{\Delta r} \right) - \left(\frac{r P \Big|_{z+\Delta z} - r P \Big|_z}{\Delta z} \right) = 0$$

Set limit $\Delta r \rightarrow 0$, limit $\Delta z \rightarrow 0$

$$\lim_{\Delta z \rightarrow 0} \left(\frac{r \rho v_z^2 \Big|_{z+\Delta z} - r \rho v_z^2 \Big|_z}{\Delta z} \right) - \lim_{\Delta r \rightarrow 0} \left(\frac{r \tau_{rz} \Big|_{r+\Delta r} - r \tau_{rz} \Big|_r}{\Delta r} \right) - \lim_{\Delta z \rightarrow 0} \left(\frac{r P \Big|_{z+\Delta z} - r P \Big|_z}{\Delta z} \right) = 0$$

$$-\frac{d(r \rho v_z^2)}{dz} - \frac{d(r \tau_{rz})}{dr} - \frac{d(r P)}{dz} = 0$$

For incompressible fluid ($\rho = \text{constant}$)

$$-\frac{\rho d(r v_z^2)}{dz} - \frac{d(r \tau_{rz})}{dr} - \frac{d(r P)}{dz} = 0$$

Since $V_z \neq f(z)$, $\frac{dv_z}{dz} = 0$ and $\frac{d(v_z^2)}{dz} = 0$.

$$-\frac{d(r \cdot \tau_{rz})}{dr} - \frac{d(r \cdot P)}{dz} = 0$$

Simplified equation by approximation $dP \approx \Delta P$ and $dz \approx \Delta z \approx L$ and $r \neq f$ (z):

$$r \frac{dP}{dz} \approx r \frac{\Delta P}{\Delta z} = \frac{r(P_L - P_0)}{L}$$

Then we obtain 1st ODE w.r.t. τ_{rz} :

$$-\frac{d(r \cdot \tau_{rz})}{dr} - \frac{r(P_L - P_0)}{L} = 0$$

Apply Newton's Law: $\tau_{rz} = -\frac{\mu dv_z}{dr}$

$$-\frac{d}{dr} \left(r \cdot \left(-\mu \frac{dv_z}{dr} \right) \right) - \frac{r(P_L - P_0)}{L} = 0$$

For isothermal system, $\mu = \text{constant}$

$$\mu \frac{d}{dr} \left(r \frac{dv_z}{dr} \right) + \frac{r(P_0 - P_L)}{L} = 0$$

$$\frac{d}{dr} \left(r \frac{dv_z}{dr} \right) = -\frac{r(P_0 - P_L)}{\mu L}$$

Perform the equation by indefinite integration (1st)

$$\int d \left(r \frac{dv_z}{dr} \right) = \int -\frac{r(P_0 - P_L)}{\mu L} dr$$

$$r \frac{dv_z}{dr} = -\frac{r^2 (P_0 - P_L)}{2\mu L} + C_1$$

$$\frac{dv_z}{dr} = -\frac{r(P_0 - P_L)}{2\mu L} + \frac{C_1}{r}$$

Perform the equation by indefinite integration (2nd)

$$\int d(v_z) = \int \frac{-r(P_0 - P_L)}{2\mu L} dr + \int \frac{C_1}{r} dr$$

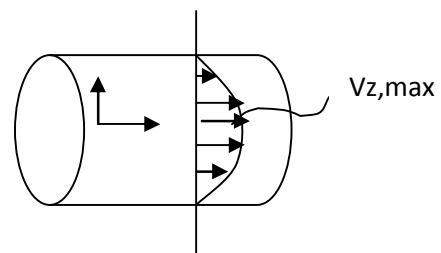
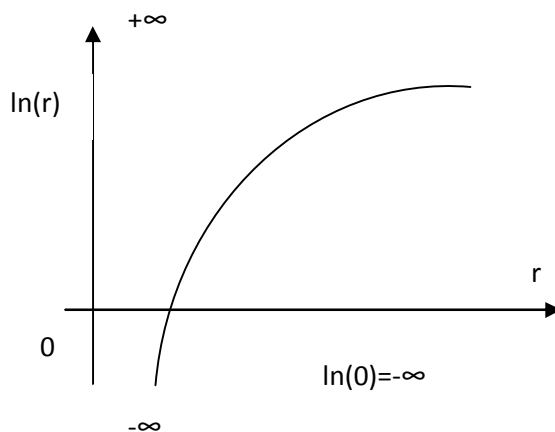
$$v_z = \frac{-r^2 (P_0 - P_L)}{4\mu L} + C_1 \ln(r) + C_2$$

Apply BC1: At $r = 0$, $V_z = V_{z, \max}$ (finite)

$$(v_{z, \max}) = -\frac{(0)^2 (P_0 - P_L)}{4\mu L} + C_1 (-\infty) + C_2$$

(finite)

$\underbrace{\hspace{2cm}}$
 $-\infty$



Note: The above equation is $-\infty$ by means of mathematic, but in fact the equation must be finite because $V_{z,\max}$ is finite. So, we force $C_1 = 0$ in order to valid the equation.

$$v_z = \frac{-r^2(P_0 - P_L)}{4\mu L} + C_1 \ln(r) + C_2$$

Finite = finite + 0 + C_2

Substitute C_1 into the equation

$$v_z = \frac{-r^2(P_0 - P_L)}{4\mu L} + C_2$$

Apply BC2: At $r=R$, $V_z=0$

$$0 = \frac{-R^2(P_0 - P_L)}{4\mu L} + C_2$$

$$C_2 = \frac{R^2(P_0 - P_L)}{4\mu L}$$

Substitute C_2 into the equation

$$v_z = \frac{-r^2(P_0 - P_L)}{4\mu L} + \frac{R^2(P_0 - P_L)}{4\mu L}$$

$$v_z = \frac{(P_0 - P_L)}{4\mu L} (R^2 - r^2)$$

So we get 1st solution as the velocity profile:

$$v_z = \frac{R^2(P_0 - P_L)}{4\mu L} \left[1 - \left(\frac{r}{R} \right)^2 \right]$$

What is expression of the maximum velocity?

Since BC2 at $r=0$, $V_z = V_{z,max}$, we get,

$$v_{z,max} = \frac{R^2(P_0 - P_L)}{4\mu L} \left[1 - \left(\frac{0}{R} \right)^2 \right]$$

$$v_{z,max} = \frac{R^2(P_0 - P_L)}{4\mu L}$$

What is equation of volumetric flow rate (Q)?

$$\int_0^Q dQ = \int_0^R (v_z \cdot 2\pi r \cdot dr)$$

$$Q = \int_0^R \frac{R^2(P_0 - P_L)}{4\mu L} \left[1 - \left(\frac{r}{R} \right)^2 \right] \cdot 2\pi r \cdot dr$$

$$Q = \frac{2\pi R^2(P_0 - P_L)}{4\mu L} \left[\frac{r^2}{2} - \frac{r^4}{4R^2} \right] \Bigg|_0^R$$

$$Q = \frac{2\pi R^2(P_0 - P_L)}{4\mu L} \left[\frac{4R^4 - 2R^4}{8R^2} \right]$$

$$Q = \frac{\pi R^4(P_0 - P_L)}{8\mu L} ; \quad (Q \propto R^4)$$

“The Hagen-Poiseuille equation”

Hydraulic Eng. (1839, German)

Physician (1841, France)

What is mean velocity (\bar{v}_z)?

$$\bar{v}_z = \frac{1}{2} v_{z,max}$$

$$\bar{v}_z = \frac{R^2(P_0 - P_L)}{8\mu L}$$

What is Force acting on surface of pipe?

$$\vec{F} = (\tau_{rz}|_{r=R})(2\pi RL)$$

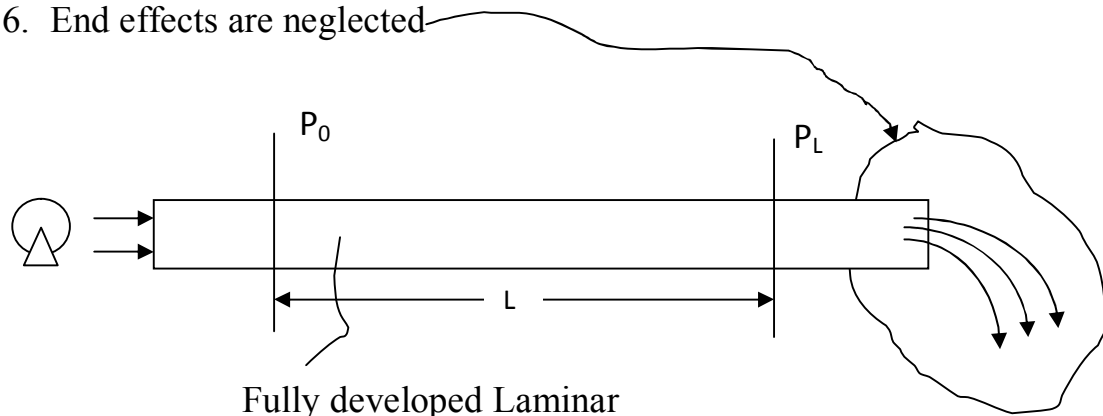
$$\vec{F} = \left(-\mu \cdot \frac{dv_z}{dr}\right)(2\pi RL)$$

$$\vec{F} = \mu \cdot \frac{R(P_0 - P_L)}{2\mu L}(2\pi RL)$$

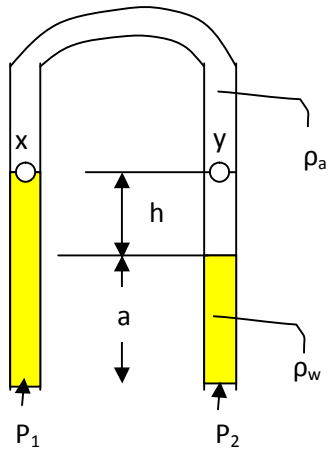
$$\vec{F} = \pi R^2(P_0 - P_L)$$

Assumptions of the Hagen-Poiseuille Equation

1. Isothermal system
2. Laminar flow
3. Incompressible fluid (v, ρ constant)
4. Newtonian fluid
5. Steady state system
6. End effects are neglected



Inverted Monometer



$$P_{\text{high}} = P_1$$

$$P_{\text{low}} = P_2$$

$$P_1 = \rho_w g h + \rho_w g a + P_x$$

$$P_2 = \rho_a g h + \rho_w g a + P_y$$

$$P_1 - P_2 = \rho_w g h + \rho_w g a + P_x - \rho_a g h - \rho_w g a - P_y$$

Because $x = y$, $P_x = P_y$

$$P_1 - P_2 = (\rho_w - \rho_a) g h$$