Example 16 Heat conduction in a cooling Fin

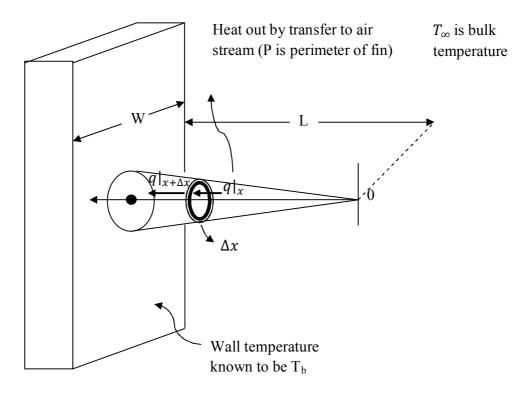


Figure 16 A simple cooling fin

Assume:

$$\frac{x}{L} = \frac{A_x}{A_L} = \frac{P_x}{P_L}$$
$$A_x = \frac{A_L}{L}x; P_x = \frac{P_L}{L}x$$

Conduction heat balance around shell at steady state

[Rate of heat in] - [Rate of heat out] - [Rate of heat loss] = 0

$$q|_{x} \cdot A_{x} - q|_{x+\Delta x} \cdot A_{x} - h(T - T_{\infty})P_{x}\Delta x = 0$$

Substitute $A_x = \frac{A_L}{L}x$ and $P_x = \frac{P_L}{L}x$ into the above equation

$$q|_{x} \cdot \frac{A_{L}}{L} x - q|_{x + \Delta x} \cdot \frac{A_{L}}{L} x - h(T - T_{\infty}) \frac{P_{L}}{L} x \Delta x = 0$$

Divided by $\frac{A_L}{L}\Delta x$ through equation and taking limit $\Delta x \to 0$ we get

$$-\lim_{\Delta x \to 0} \left(\frac{q|_{x+\Delta x} \cdot x - q|_x \cdot x}{\Delta x} \right) - h(T - T_{\infty}) \frac{P_L}{A_L} x = 0$$

From the definition of derivative equation we obtain

$$-\frac{\mathrm{d}(x \cdot q_x)}{\mathrm{d}x} - h(T - T_\infty)\frac{P_L}{A_L}x = 0$$
$$-\frac{\mathrm{d}(x \cdot q_x)}{\mathrm{d}x} = h(T - T_\infty)\frac{P_L}{A_L}x$$

Apply Fourier's law;

$$\frac{k}{x}\frac{d}{dx}\left(x\frac{dT}{dx}\right) = h(T - T_{\infty})\frac{P_{L}}{A_{L}}$$
$$\frac{1}{x}\frac{d}{dx}\frac{xdT}{dx} = (T - T_{\infty})\frac{P_{L}h}{kA_{L}}$$
(2)

Set dimensionless group: $\theta = \frac{(T - T_{\infty})}{(T_b - T_{\infty})}$: $dT = (T_b - T_{\infty})d\theta$, equation (2) becomes:

$$d\theta = d \left[\frac{(T - T_{\infty})}{(T_b - T_{\infty})} \right]$$
$$d\theta = \frac{1}{(T_b - T_{\infty})} d[(T - T_{\infty})]$$
$$d\theta = \frac{1}{(T_b - T_{\infty})} [dT + 0]$$
$$(T_b - T_{\infty}) d\theta = dT$$

$$\frac{1}{x}\frac{d}{dx}\left(\frac{xd\theta}{dx}\right) = \frac{(T-T_{\infty})}{(T_b - T_{\infty})}\frac{P_Lh}{A_Lk}$$
$$\frac{1}{x}\frac{d}{dx}\left(\frac{xd\theta}{dx}\right) = \theta\frac{P_Lh}{A_Lk}$$
$$\frac{d}{dx}\left(x\cdot\frac{d\theta}{dx}\right) = \theta x\frac{P_Lh}{A_Lk}$$
$$x\frac{d^2\theta}{dx^2} + \frac{d\theta}{dx} - \theta x\left(\frac{P_Lh}{A_Lk}\right) = 0$$
$$x^2\frac{d^2\theta}{dx^2} + x\frac{d\theta}{dx} - x^2\theta\left(\frac{P_Lh}{A_Lk}\right) = 0$$

Set: $\frac{P_L h}{A_L k} = \beta^2$;

$$x^{2}\frac{d^{2}\theta}{dx^{2}} + x\frac{d\theta}{dx} - x^{2}\beta^{2}\theta = 0$$

$$x^2 \frac{d^2\theta}{dx^2} + x \frac{d\theta}{dx} - (x^2\beta^2 - 0^2)\theta = 0$$
(3)

From Modified Bessel:

$$x^{2}\frac{d^{2}\theta}{dx^{2}} + x\frac{d\theta}{dx} - (x^{2}\beta^{2} + n^{2})\theta = 0$$

Solution

$$\theta = AI_0(\beta x) + BK_0(\beta x) \tag{4}$$

<u>Case I</u>: The fin is very long, and the temperature at the end of the fin is essentially that of the surrounding fluid.

Assume long fin; $L \rightarrow \infty$

BC1:
$$x = 0$$
; $\frac{dT}{dx}\Big|_{x=0} = 0$: $\frac{d\theta}{dx}\Big|_{x=0} = 0$
BC2: $x = L$; $T = T_b$: $\theta = 1$

$$\frac{d\theta}{dx} = A \frac{dI_0(\beta x)}{dx} + \frac{BdK_0(\beta x)}{dx}$$
$$\frac{d\theta}{dx} = A\beta I_1(\beta x) - BK_1\beta(\beta x)$$

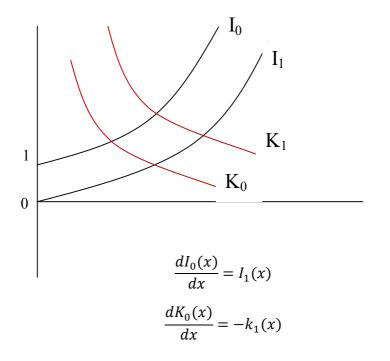
Use BC1 to substitute into eqn (3);

$$0 = AI_1(0) - BK_1(0)$$

Finite = 0 ∞

Because term $K_1(0) = \infty$ and the equation is finite, we have to force B = 0 to valid the equation. Eqn (4) becomes:

$$\theta = AI_0(\beta x) \tag{5}$$



Use BC2 substitute into equation (5)

$$1 = AI_0(\beta L)$$
$$A = \frac{1}{I_0(\beta L)}$$
$$\theta = \frac{I_0(\beta x)}{I_0(\beta L)}$$

Thus, we get the temperature profile:

$$T = T_{\infty} + (T_b - T_{\infty}) \frac{I_0(\beta x)}{I_0(\beta L)}; \ \beta = \frac{P_L h}{A_L k}$$

We can determine heat loss by performing derivatives with respect to x of temperature profile

$$Q_{loss} = A(-k) \frac{dT}{dx} \Big|_{x=L}$$
$$\frac{dT}{dx} \Big|_{x=L} = \frac{(T_b - T_\infty)}{I_0(\beta L)} \beta I_1(\beta x) \Big|_{x=L}$$

$$Q_{loss} = A(-k) \frac{dT}{dx} \Big|_{x=L} = \beta (T_b - T_\infty) \frac{I_1(\beta x)|_{x=L}}{I_0(\beta L)} (-k) (A_L)$$
$$Q_{loss} = -kA_L (T_b - T_\infty) \beta \frac{I_1(\beta L)}{I_0(\beta L)}$$
$$Q_{ideal} = h (T_b - T_\infty) \pi R \sqrt{R^2 + L^2}$$
$$Q_{no \ fin} = (T_b - T_\infty) \pi R_b^2$$

We can determine *fin efficiency* (η_f) which is defined by

$$\eta_f = \frac{\text{actual rate of heat loss from the fin}}{\text{rate of heat loss from an isothermal fin at } T_w}$$

$$\eta_f = \frac{Q_{loss}}{Q_{ideal}}$$

We can determine *fin effectiveness* (ε_f)

$$\varepsilon_f = rac{Q_{actual}}{Q_{no\ fin}}$$

<u>Case II</u>: The end of fin is insulated so that $\frac{dt}{dx} = 0$ at x=L (No heat is lost from the end or from the edges.)

BC1:
$$x = x_1 : \frac{dT}{dx}\Big|_{x=x_1}; \frac{d\theta}{dx}\Big|_{x=x_1} = 0$$

BC2: $x = L: T = T_b; \theta = 1$

<u>Case III</u>: The fin is of finite length and loses heat by convection from its end. (Facing convection BC at the tip of fin)

BC1:
$$q|_{x=x_1} = -\frac{kdT}{dx}\Big|_{x=x_1} = h(T_L - T_\infty)$$

 $-(T_b - T_\infty)\frac{d\theta}{dx}\Big|_{x=x_1} = h(T_L - T_\infty)$
 $-k\frac{d\theta}{dx}\Big|_{x=x_1} = \frac{h(T|_{x=x_1} - T_\infty)}{T_b - T_\infty}$
 $\frac{d\theta}{dx}\Big|_{x=x_1} = \frac{-h\emptyset|_{x=x_1}}{k}$
BC2: $x = L$: $T = T_b$; $\theta = 1$

